

## Thermo 1 (MEP 261)

### *Thermodynamics An Engineering Approach*

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### Sheet 3:Chapter 3

3-26 Complete this table for H<sub>2</sub>O:

$T, ^\circ\text{C}$	$P, \text{kPa}$	$v, \text{m}^3/\text{kg}$	Phase description
50		4.16	
	200		Saturated vapor
250	400		
110	600		

3-26 Complete the following table for H<sub>2</sub>O:

$T, ^\circ\text{C}$	$P, \text{kPa}$	$v, \text{m}^3/\text{kg}$	Phase description
50	12.352	4.16	Saturated mixture
120.21	200	0.8858	Saturated vapor
250	400	0.5952	Superheated vapor
110	600	0.001051	Compressed liquid

**3-27** Reconsider Prob. 3-26. Using EES (or other) software, determine the missing properties of water. Repeat the solution for refrigerant-134a, refrigerant-22, and ammonia.

**3-30** Complete this table for H<sub>2</sub>O:

$T, ^\circ\text{C}$	$P, \text{kPa}$	$h, \text{kJ/kg}$	$x$	Phase description
	200		0.7	
140		1800		
	950		0.0	
80	500			
	800	3162.2		

**Solution** Complete the following table for H<sub>2</sub>O:

$T, ^\circ\text{C}$	$P, \text{kPa}$	$h, \text{kJ/kg}$	$x$	Phase description
<i>120.21</i>	200	<i>2045.8</i>	0.7	<i>Saturated mixture</i>
140	<i>361.53</i>	1800	<i>0.565</i>	<i>Saturated mixture</i>
<i>177.66</i>	950	<i>752.74</i>	0.0	<i>Saturated liquid</i>
80	500	<i>335.37</i>	---	<i>Compressed liquid</i>
<i>350.0</i>	800	3162.2	---	<i>Superheated vapor</i>

**3-31** Complete this table for refrigerant-134a:

$T, ^\circ\text{C}$	$P, \text{kPa}$	$v, \text{m}^3/\text{kg}$	Phase description
-8	320		
30		0.015	
	180		Saturated vapor
80	600		

**Solution** Complete the following table for Refrigerant-134a:

T, °C	P, kPa	$\nu$ , m <sup>3</sup> / kg	Phase description
-8	320	<i>0.0007569</i>	<i>Compressed liquid</i>
30	<i>770.64</i>	0.015	<i>Saturated mixture</i>
<i>-12.73</i>	180	<i>0.11041</i>	Saturated vapor
80	600	<i>0.044710</i>	<i>Superheated vapor</i>

**3-32** Complete this table for refrigerant-134a:

T, °C	P, kPa	u, kJ/kg	Phase description
20		95	
-12			Saturated liquid
	400	300	
8	600		

**Solution** Complete the following table for Refrigerant-134a:

T, °C	P, kPa	u, kJ / kg	Phase description
20	<i>572.07</i>	95	<i>Saturated mixture</i>
-12	<i>185.37</i>	<i>35.78</i>	Saturated liquid
<i>86.24</i>	400	300	<i>Superheated vapor</i>
8	600	<i>62.26</i>	<i>Compressed liquid</i>

**3-34** Complete this table for H<sub>2</sub>O:

T, °C	P, kPa	$\nu$ , m <sup>3</sup> /kg	Phase description
140		0.05	
	550		Saturated liquid
125	750		
500		0.140	

**Solution** Complete the following table for H<sub>2</sub>O:

T, °C	P, kPa	$\nu$ , m <sup>3</sup> / kg	Phase description
140	<i>361.53</i>	0.05	<i>Saturated mixture</i>
<i>155.46</i>	550	<i>0.001097</i>	Saturated liquid
125	750	<i>0.001065</i>	<i>Compressed liquid</i>
500	<i>2500</i>	0.140	<i>Superheated vapor</i>

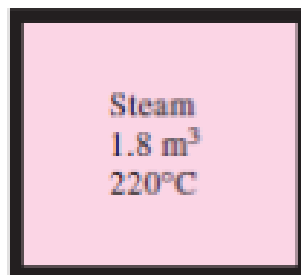
**3–35** Complete this table for H<sub>2</sub>O:

$T, ^\circ\text{C}$	$P, \text{kPa}$	$u, \text{kJ/kg}$	Phase description
	400	1450	
220			Saturated vapor
190	2500		
	4000	3040	

**Solution** Complete the following table for H<sub>2</sub>O:

$T, ^\circ\text{C}$	$P, \text{kPa}$	$u, \text{kJ/kg}$	Phase description
<i>143.61</i>	400	1450	<i>Saturated mixture</i>
220	<i>2319.6</i>	<i>2601.3</i>	Saturated vapor
190	2500	<i>805.15</i>	<i>Compressed liquid</i>
<i>466.21</i>	4000	3040	<i>Superheated vapor</i>

**3–36** A 1.8-m<sup>3</sup> rigid tank contains steam at 220°C. One-third of the volume is in the liquid phase and the rest is in the vapor form. Determine (a) the pressure of the steam, (b) the quality of the saturated mixture, and (c) the density of the mixture.



**FIGURE P3–36**

**Solution** A rigid tank contains steam at a specified state. The pressure, quality, and density of steam are to be determined.

**Properties** At 220°C  $\nu_f = 0.001190 \text{ m}^3/\text{kg}$  and  $\nu_g = 0.08609 \text{ m}^3/\text{kg}$  (Table A-4).

**Analysis** (a) Two phases coexist in equilibrium, thus we have a saturated liquid-vapor mixture. If the pressure of the steam is the saturation pressure at the given temperature, then the pressure in the tank must be the saturation pressure at the specified temperature,

$$P = T_{\text{sat}@220^\circ\text{C}} = \mathbf{2320 \text{ kPa}}$$

(b) The total mass and the quality are determined as

$$m_f = \frac{V_f}{\nu_f} = \frac{1/3 \times (1.8 \text{ m}^3)}{0.001190 \text{ m}^3/\text{kg}} = 504.2 \text{ kg}$$

$$m_g = \frac{V_g}{\nu_g} = \frac{2/3 \times (1.8 \text{ m}^3)}{0.08609 \text{ m}^3/\text{kg}} = 13.94 \text{ kg}$$

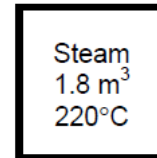
$$m_t = m_f + m_g = 504.2 + 13.94 = 518.1 \text{ kg}$$

$$x = \frac{m_g}{m_t} = \frac{13.94}{518.1} = \mathbf{0.0269}$$

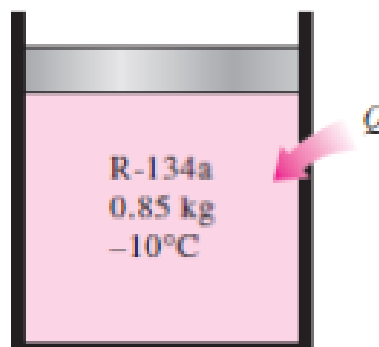
(c) The density is determined from

$$\nu = \nu_f + x(\nu_g - \nu_f) = 0.001190 + (0.0269)(0.08609) = 0.003474 \text{ m}^3/\text{kg}$$

$$\rho = \frac{1}{\nu} = \frac{1}{0.003474} = \mathbf{287.8 \text{ kg/m}^3}$$



**3–37** A piston–cylinder device contains 0.85 kg of refrigerant-134a at  $-10^\circ\text{C}$ . The piston that is free to move has a mass of 12 kg and a diameter of 25 cm. The local atmospheric pressure is 88 kPa. Now, heat is transferred to refrigerant-134a



**FIGURE P3–37**

**Solution** A piston-cylinder device contains R-134a at a specified state. Heat is transferred to R-134a. The final pressure, the volume change of the cylinder, and the enthalpy change are to be determined.

**Analysis** (a) The final pressure is equal to the initial pressure, which is determined from

$$P_2 = P_1 = P_{\text{atm}} + \frac{m_p g}{\pi D^2/4} = 88 \text{ kPa} + \frac{(12 \text{ kg})(9.81 \text{ m/s}^2)}{\pi(0.25 \text{ m})^2/4} \left( \frac{1 \text{ kN}}{1000 \text{ kg}\cdot\text{m/s}^2} \right) = \mathbf{90.4 \text{ kPa}}$$

(b) The specific volume and enthalpy of R-134a at the initial state of 90.4 kPa and  $-10^\circ\text{C}$  and at the final state of 90.4 kPa and  $15^\circ\text{C}$  are (from EES)

$$\begin{aligned} v_1 &= 0.2302 \text{ m}^3/\text{kg} & h_1 &= 247.76 \text{ kJ/kg} \\ v_2 &= 0.2544 \text{ m}^3/\text{kg} & h_2 &= 268.16 \text{ kJ/kg} \end{aligned}$$

The initial and the final volumes and the volume change are

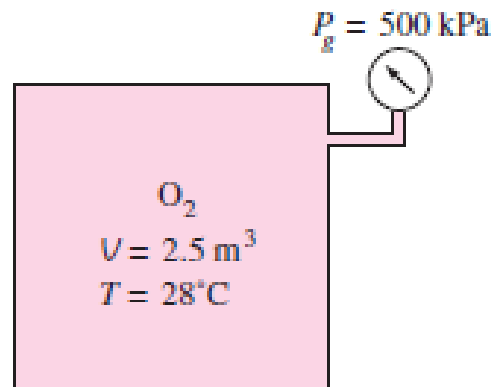
$$\begin{aligned} V_1 &= m v_1 = (0.85 \text{ kg})(0.2302 \text{ m}^3/\text{kg}) = 0.1957 \text{ m}^3 \\ V_2 &= m v_2 = (0.85 \text{ kg})(0.2544 \text{ m}^3/\text{kg}) = 0.2162 \text{ m}^3 \\ \Delta V &= 0.2162 - 0.1957 = \mathbf{0.0205 \text{ m}^3} \end{aligned}$$

(c) The total enthalpy change is determined from

$$\Delta H = m(h_2 - h_1) = (0.85 \text{ kg})(268.16 - 247.76) \text{ kJ/kg} = \mathbf{17.4 \text{ kJ/kg}}$$



**3-77** The pressure gage on a  $2.5\text{-m}^3$  oxygen tank reads 500 kPa. Determine the amount of oxygen in the tank if the temperature is  $28^\circ\text{C}$  and the atmospheric pressure is 97 kPa.



**FIGURE P3-77**

**Solution** The pressure and temperature of oxygen gas in a storage tank are given. The mass of oxygen in the tank is to be determined.

**Assumption** At specified conditions, oxygen behaves as an ideal gas.

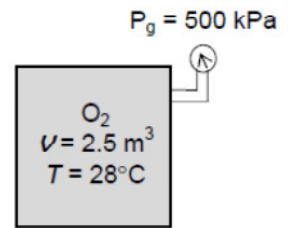
**Properties** The gas constant of oxygen is  $R = 0.2598 \text{ kPa}\cdot\text{m}^3/\text{kg}\cdot\text{K}$  (Table A-1).

**Analysis** The absolute pressure of  $\text{O}_2$  is

$$P = P_g + P_{\text{atm}} = 500 + 97 = 597 \text{ kPa}$$

Treating  $\text{O}_2$  as an ideal gas, the mass of  $\text{O}_2$  in tank is determined to be

$$m = \frac{PV}{RT} = \frac{(597 \text{ kPa})(2.5 \text{ m}^3)}{(0.2598 \text{ kPa}\cdot\text{m}^3/\text{kg}\cdot\text{K})(28+273)\text{K}} = \mathbf{19.08 \text{ kg}}$$



**3–80** A  $1\text{-m}^3$  tank containing air at  $25^\circ\text{C}$  and  $500 \text{ kPa}$  is connected through a valve to another tank containing  $5 \text{ kg}$  of air at  $35^\circ\text{C}$  and  $200 \text{ kPa}$ . Now the valve is opened, and the entire system is allowed to reach thermal equilibrium with the surroundings, which are at  $20^\circ\text{C}$ . Determine the volume of the second tank and the final equilibrium pressure of air.

**Answers:**  $2.21 \text{ m}^3$ ,  $284.1 \text{ kPa}$

**Solution** Two rigid tanks connected by a valve to each other contain air at specified conditions. The volume of the second tank and the final equilibrium pressure when the valve is opened are to be determined.

**Assumptions** At specified conditions, air behaves as an ideal gas.

**Properties** The gas constant of air is  $R = 0.287 \text{ kPa}\cdot\text{m}^3/\text{kg}\cdot\text{K}$  (Table A-1).

**Analysis** Let's call the first and the second tanks A and B. Treating air as an ideal gas, the volume of the second tank and the mass of air in the first tank are determined to be

$$V_B = \left( \frac{m_1 R T_1}{P_1} \right)_B = \frac{(5 \text{ kg})(0.287 \text{ kPa}\cdot\text{m}^3/\text{kg}\cdot\text{K})(308 \text{ K})}{200 \text{ kPa}} = \mathbf{2.21 \text{ m}^3}$$

$$m_A = \left( \frac{P_1 V}{R T_1} \right)_A = \frac{(500 \text{ kPa})(1.0 \text{ m}^3)}{(0.287 \text{ kPa}\cdot\text{m}^3/\text{kg}\cdot\text{K})(298 \text{ K})} = 5.846 \text{ kg}$$

Thus,

$$V = V_A + V_B = 1.0 + 2.21 = 3.21 \text{ m}^3$$

$$m = m_A + m_B = 5.846 + 5.0 = 10.846 \text{ kg}$$

Then the final equilibrium pressure becomes

$$P_2 = \frac{m R T_2}{V} = \frac{(10.846 \text{ kg})(0.287 \text{ kPa}\cdot\text{m}^3/\text{kg}\cdot\text{K})(293 \text{ K})}{3.21 \text{ m}^3} = \mathbf{284.1 \text{ kPa}}$$

